

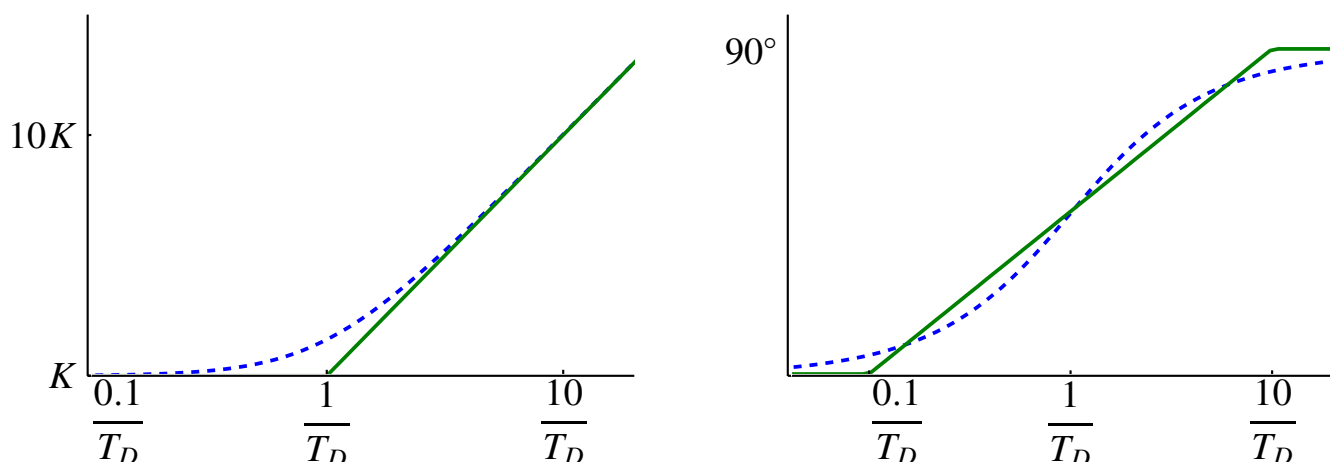
# FREQUENCY-RESPONSE DESIGN

## 9.1: PD and lead compensation networks

- The frequency-response methods we have seen so far largely tell us about stability and stability margins of a closed-loop system based on open-loop response.
- Now, we look at frequency-response based design methods which primarily aim at improving stability margins.
- Also, given the relationship between  $PM$  and performance, we have some idea of transient response as well.
- Start thinking of Bode-magnitude and Bode-phase plots as “LEGO” to make the frequency response we want.

### PD compensation

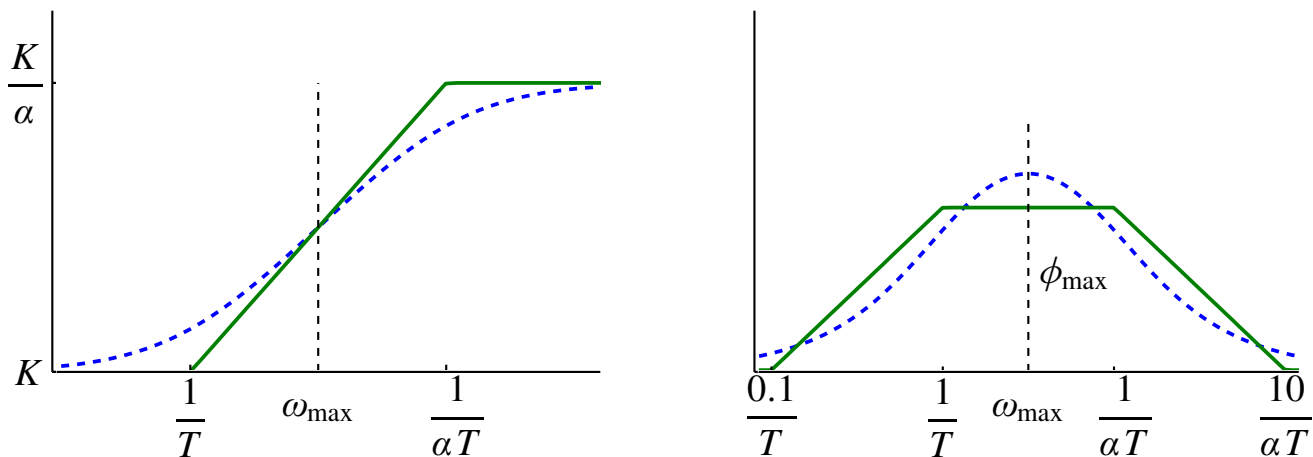
- Compensator  $D(s) = K(1 + T_D s)$ .
- We have seen from root-locus that this has a stabilizing effect.
- Magnitude and phase effect:



- PD controller increases phase for frequencies over  $\omega \approx 0.1/T_D$ . So, locate  $1/T_D < \text{crossover}$  so that phase margin at crossover is better.
- Problem: Magnitude response continues to increase as frequency increases. This amplifies high-frequency sensor noise.

### Lead compensation

- Compensator  $D(s) = K \left[ \frac{Ts + 1}{\alpha Ts + 1} \right]$ ,  $\alpha < 1$ .
- Approximate PD control for frequencies up to  $\omega = \frac{1}{\alpha T}$ .
- Magnitude and phase effect:



- The phase contributed at frequency  $\omega$  is

$$\phi = \tan^{-1}(T\omega) - \tan^{-1}(\alpha T\omega).$$

- To find where the maximum phase contribution is, we take the derivative of  $\phi$  with respect to  $\omega$  and set it to zero

$$\begin{aligned} \frac{d\phi}{d\omega} &= \frac{d \tan^{-1}(T\omega)}{d\omega} - \frac{d \tan^{-1}(\alpha T\omega)}{d\omega} \\ &= \frac{T}{T^2\omega^2 + 1} - \frac{\alpha T}{\alpha^2 T^2\omega^2 + 1} = 0 \end{aligned}$$

$$\alpha^2 T^3 \omega^2 + T = \alpha T^3 \omega^2 + \alpha T$$

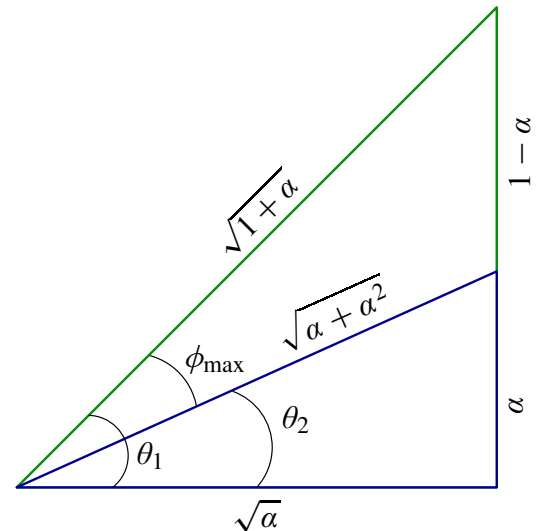
$$T^3 \omega^2 \alpha (\alpha - 1) = T (\alpha - 1)$$

$$\omega_{\max} = \frac{1}{T \sqrt{\alpha}}$$

- For this frequency, the amount of phase added is

$$\begin{aligned} \phi_{\max} &= \tan^{-1}(T \omega_{\max}) - \tan^{-1}(\alpha T \omega_{\max}) \\ &= \underbrace{\tan^{-1}\left(\frac{1}{\sqrt{\alpha}}\right)}_{\theta_1} - \underbrace{\tan^{-1}\left(\frac{\alpha}{\sqrt{\alpha}}\right)}_{\theta_2} \end{aligned}$$

To get a simpler expression, consider the triangles to the right.



- We can solve for  $\phi_{\max}$  using the law of cosines. This gives:

$$(1 - \alpha)^2 = (1 + \alpha) + (\alpha + \alpha^2) - 2\sqrt{1 + \alpha}\sqrt{\alpha + \alpha^2} \cos(\phi_{\max})$$

$$1 - 2\alpha + \alpha^2 = 1 + 2\alpha + \alpha^2 - 2\sqrt{1 + \alpha}\sqrt{\alpha + \alpha^2} \cos(\phi_{\max})$$

$$\cos(\phi_{\max}) = \frac{4\alpha}{2\sqrt{1 + \alpha}\sqrt{\alpha + \alpha^2}} = \frac{2\sqrt{\alpha}}{(1 + \alpha)}$$

- Now, we remember that  $\cos^2(\theta) + \sin^2(\theta) = 1$ , so

$$\begin{aligned} \sin^2(\phi_{\max}) &= 1 - \frac{4\alpha}{(1 + \alpha)^2} \\ &= \frac{(1 + \alpha)^2 - 4\alpha}{(1 + \alpha)^2} \\ &= \frac{(1 - \alpha)^2}{(1 + \alpha)^2} \end{aligned}$$

- So, we conclude that

$$\phi_{\max} = \sin^{-1} \left( \frac{1 - \alpha}{1 + \alpha} \right).$$

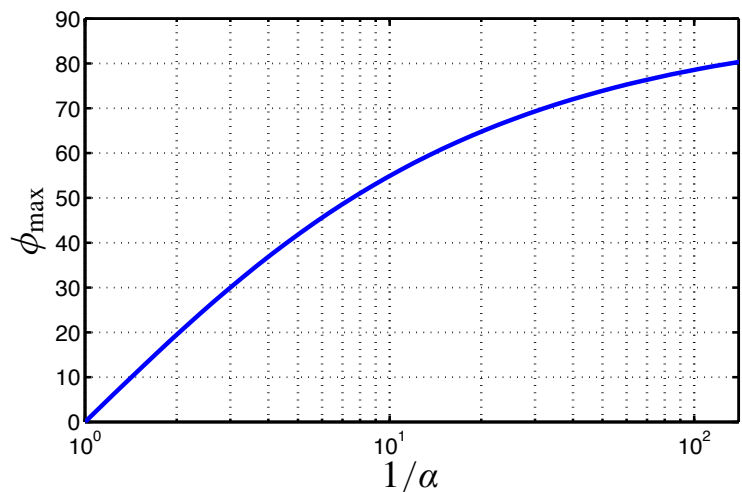
- We can invert this to find  $\alpha$  to add a certain phase  $\phi_{\max}$

$$\alpha = \frac{1 - \sin(\phi_{\max})}{1 + \sin(\phi_{\max})}.$$

- We can show that  $\omega_{\max}$  occurs mid-way between  $\frac{1}{T}$  and  $\frac{1}{\alpha T}$  on a log scale

$$\begin{aligned} \log(\omega_{\max}) &= \log \left( \frac{\frac{1}{\sqrt{T}}}{\sqrt{\alpha T}} \right) = \log \left( \frac{1}{\sqrt{T}} \right) + \log \left( \frac{1}{\sqrt{\alpha T}} \right) \\ &= \frac{1}{2} \left[ \log \left( \frac{1}{T} \right) + \log \left( \frac{1}{\alpha T} \right) \right]. \end{aligned}$$

- How much can we improve phase with a lead network?



- If we need more phase improvement, we can use a double-lead compensator:

$$D(s) = K \left[ \frac{Ts + 1}{\alpha Ts + 1} \right]^2.$$

- Need to compromise between good phase margin and good sensor noise rejection at high frequency.

## 9.2: Design method #1 for lead controllers

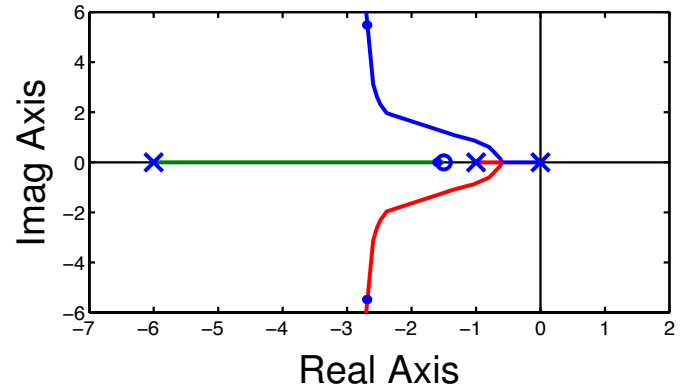
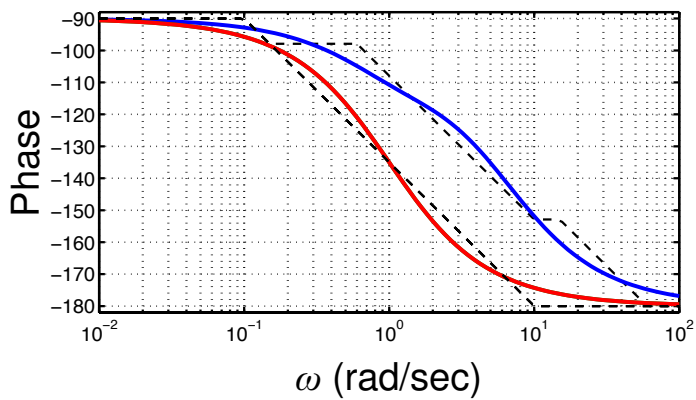
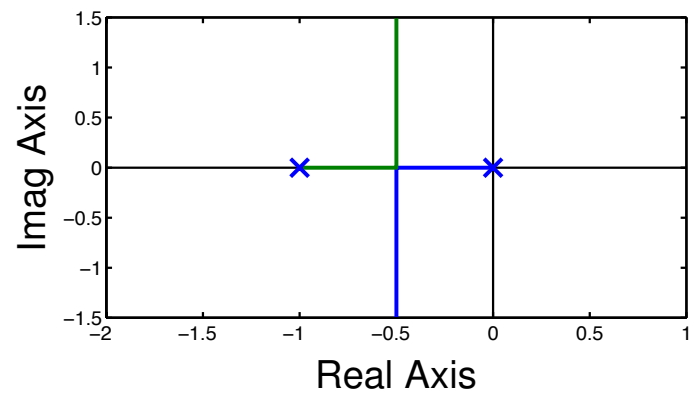
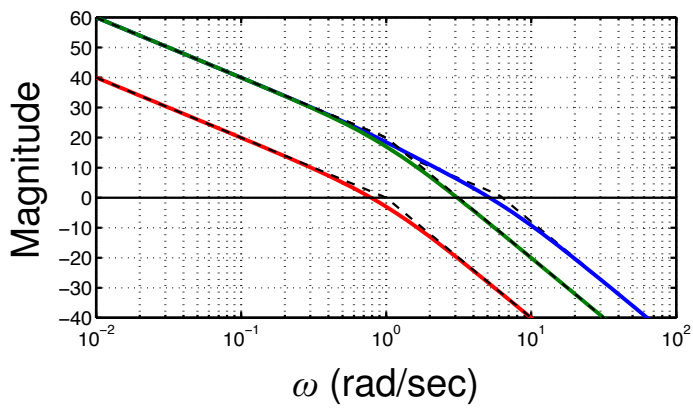
- Design specification include:
  - Desired steady-state error.
  - Desired phase margin.
- Compute steady-state error of compensated system. Set equal to desired steady-state error by computing  $K$ .
- Evaluate the  $PM$  of the uncompensated system using the value of  $K$  from above.  $\omega_{\max}$  initially set to crossover frequency.
- Add a small amount of  $PM$  ( $5^\circ$  to  $12^\circ$ ) to the needed phase lead. (Lead network moves crossover point, so need to design conservatively): Gives  $\phi_{\max}$ .
- Determine  $\alpha = \frac{1 - \sin(\phi_{\max})}{1 + \sin(\phi_{\max})}$ .
- Determine  $T = \frac{1}{\omega_{\max} \sqrt{\alpha}}$ .
- Iterate if necessary (choose a slightly different  $\omega_{\max}$  or  $\phi_{\max}$ ).

**EXAMPLE:** Plant  $G(s) = \frac{1}{s(s+1)}$

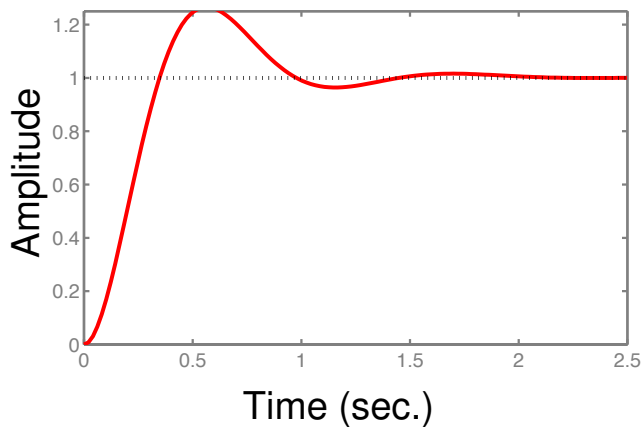
- Specification 1: Want steady-state error for ramp input  $< 0.1$ .
- Specification 2: Want overshoot  $M_p < 25\%$ .
- Start by computing steady-state error

$$\begin{aligned}
 e_{ss} &= \lim_{s \rightarrow 0} s \left[ \frac{1}{1 + D(s)G(s)} \right] R(s) \\
 &= \lim_{s \rightarrow 0} \left[ \frac{1}{s + D(s)\frac{1}{(s+1)}} \right] = \frac{1}{D(0)}.
 \end{aligned}$$

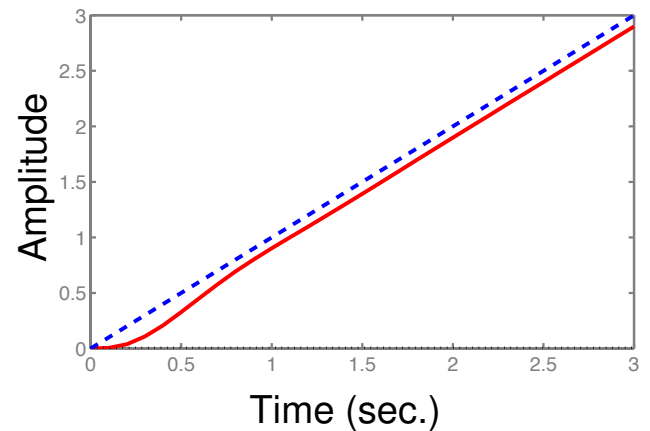
- $D(s) = K \left[ \frac{Ts + 1}{\alpha Ts + 1} \right]$  so  $D(0) = K$ .
- So,  $\frac{1}{D(0)} = \frac{1}{K} = 0.1 \quad \dots \quad K = 10$ .
- In Bode plot below, red line is original plant. Green line is plant with added gain of  $K = 10$ , but no lead compensator.
- We see that the new crossover is at  $3 \text{ rad s}^{-1}$ , so  $\omega_{\max} = 3 \text{ rad s}^{-1}$ .
- Overshoot specification  $M_p < 25\%$  gives  $PM > 45^\circ$ .
- Evaluating (red line) Bode diagram of  $KG(s)$  at crossover, we see we have a phase margin of  $18^\circ$ . Need about  $27^\circ$  more to meet spec.
- Note, addition of pole and zero from lead network will shift crossover frequency somewhat. To be safe, design for added phase of  $37^\circ$  instead of  $27^\circ$ .
- $\alpha = \frac{1 - \sin(37^\circ)}{1 + \sin(37^\circ)} = 0.25. \quad T = \frac{1}{\sqrt{0.25}(3)} = 0.667$ .
- So,  $D(s) = 10 \left[ \frac{2s/3 + 1}{2s/12 + 1} \right]$ .
- In plots below, the blue line is the Bode plot of the compensated system  $KD(s)G(s)$ .
  - Also, uncompensated root locus is plotted top-right; compensated root locus is plotted bottom-right.
- We find that we just missed specifications:  $PM = 44^\circ$ . Iterate if desired to meet specs.



Step Response



Ramp Response



### ■ Summary of design example.

1. Determine dc-gain so that steady-state errors meet spec.
2. Select  $\alpha$  and  $T$  to achieve an acceptable  $PM$  at crossover.

## 9.3: Design method #2 for lead controllers

- Design specifications
  - Desired closed-loop bandwidth requirements (rather than  $e_{ss}$ ).
  - Desired  $PM$  requirements.
- Choose open-loop crossover frequency  $\omega_c$  to be half the desired closed-loop bandwidth.
- Evaluate  $K_G = |G(j\omega_c)|$  and  $\phi_G = \angle G(j\omega_c)$ .
- Compute phase lead required  $\phi_{\max} = PM - \phi_G - 180$ .
- Compute  $\alpha = \frac{1 - \sin(\phi_{\max})}{1 + \sin(\phi_{\max})}$ .
- Compute  $T = \frac{1}{\sqrt{\alpha}\omega_c}$ .
- Compensation is

$$D(s) = K \left[ \frac{Ts + 1}{\alpha Ts + 1} \right]$$

$$|D(j\omega_c)| = K \sqrt{\frac{\frac{1}{\alpha} + 1}{\alpha + 1}} = K \frac{\sqrt{\alpha}}{\sqrt{\alpha}} \sqrt{\frac{\frac{1}{\alpha} + 1}{\alpha + 1}} = \frac{K}{\sqrt{\alpha}}$$

- Open-loop gain at  $\omega_c$  is designed to be 1:

$$|D(j\omega_c)||G(j\omega_c)| = 1$$

$$\frac{K}{\sqrt{\alpha}}|G(j\omega_c)| = 1$$

$$K = \frac{\sqrt{\alpha}}{K_G}$$

- Our design is now complete.



**EXAMPLE:** Desired gain crossover frequency of 10 radians per second.

Desired  $PM$  of  $60^\circ$ .  $G(s) = \frac{1}{s(s+1)}$ .

$$\blacksquare K_G = \left| \frac{1}{j10(j10+1)} \right| = 0.01, \quad \phi_G = -174.3^\circ.$$

$$\blacksquare \phi_{\max} = 60^\circ + 174.3^\circ - 180^\circ = 54.3^\circ.$$

$$\blacksquare \alpha = \frac{1 - \sin(\phi_{\max})}{1 + \sin(\phi_{\max})} = 0.1037.$$

$$\blacksquare T = \frac{1}{\sqrt{\alpha}10} = 0.3105.$$

$$\blacksquare K_D = \frac{\sqrt{\alpha}}{K_G} = 32.36.$$

■ Matlab code to automate this procedure:

```
% [K,T,alpha]=bod_lead(np,dp,wc,PM)
% Computes the lead compensation of the plant np/dp to have
% phase margin PM (in degrees) at crossover frequency wc (in radians/sec).
% e.g., [K,T,alpha]=bod_lead([1],[1 1 0],5,60);
function [K,T,alpha]=bod_lead(np,dp,wc,PM)

% first compute the plant response at wc.
[magc,phc]=bode(np,dp,wc);

% now, compute the needed phase lead at wc and convert to radians
% for use with "sine"
phir=(-180+PM-phc)*pi/180;

if abs(phir)>pi/2,
    fprintf('A simple phase lead/lag cannot change the phase by more\n');
    fprintf('than +/- 90 degrees.\n');
    error('Aborting. ');
end;

% Compute alpha, T, and K for compensator
alpha=(1-sin(phir))/(1+sin(phir));
T=1/(wc*sqrt(alpha)); K=sqrt(alpha)/magc;
```

```

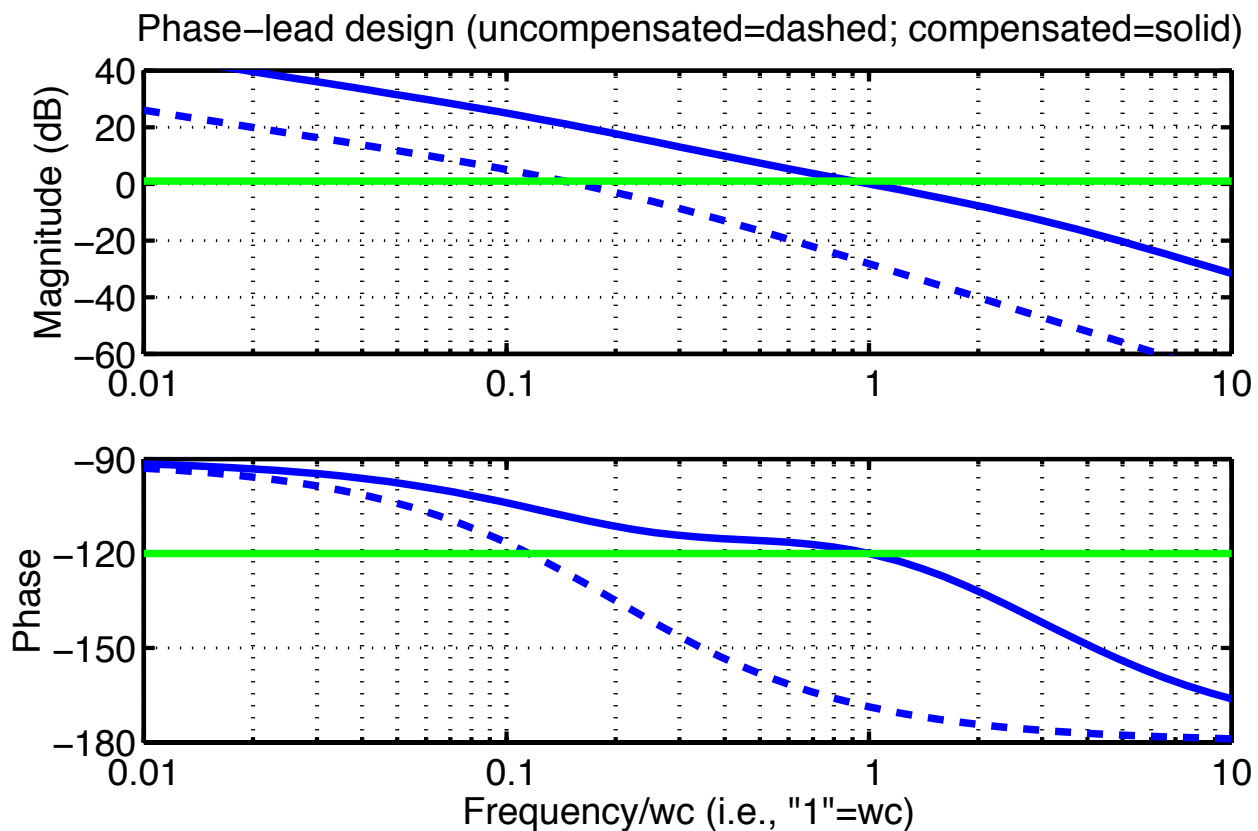
% compute the new open-loop system by convolving the plant polynomials
% with the compensator polynomials.
nol=conv(np,K*[T 1]); dol=conv(dp,[alpha*T 1]);

% check the solution by plotting the Bode plot for the new open-loop
% polynomials. Include the frequency w=1
% to get the full resonance response to show the gain margin. Also, plot
% the uncompensated Bode response.
w=logspace(-2,1)*wc; w(34)=wc; clf;
[mag1,ph1]=bode(np,dp,w); [mag2,ph2]=bode(nol,dol,w);
subplot(211);
semilogx(w/wc,20*log10(mag1),'--'); hold on;
semilogx(w/wc,20*log10(mag2)); grid; plot(w/wc,0*w+1,'g-');
ylabel('Magnitude (dB)');
title('Phase-lead design (uncompensated=dashed; compensated=solid)');

subplot(212);
semilogx(w/wc,ph1,'--'); hold on; semilogx(w/wc,ph2); grid;
plot(w/wc,0*w-180+PM,'g-');
ylabel('Phase'); xlabel('Frequency/wc (i.e., "1"=wc)');

```

## ■ Matlab results:

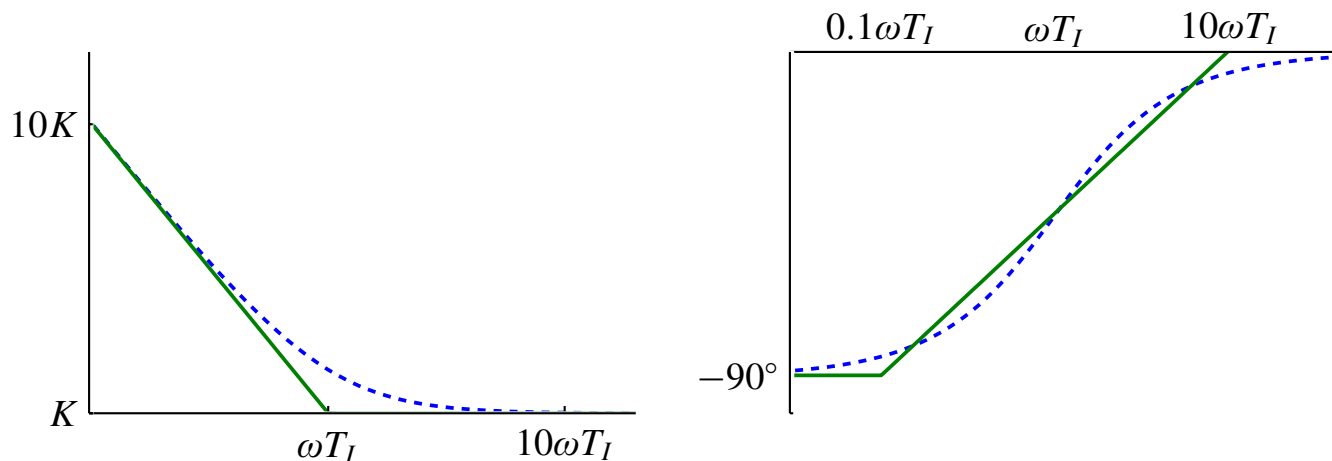


## 9.4: PI and lag compensation

### PI compensation

- In many problems it is important to keep bandwidth low, and also reduce steady-state error.
- PI compensation used here.

$$D(s) = K \left[ 1 + \frac{1}{T_I s} \right].$$



- Infinite gain at zero frequency
  - Reduces steady-state error to step, ramp, etc.
  - But also has integrator “anti-windup” problems.
- Adds phase below breakpoint.
  - We want to keep breakpoint frequency very low to keep from destabilizing system.

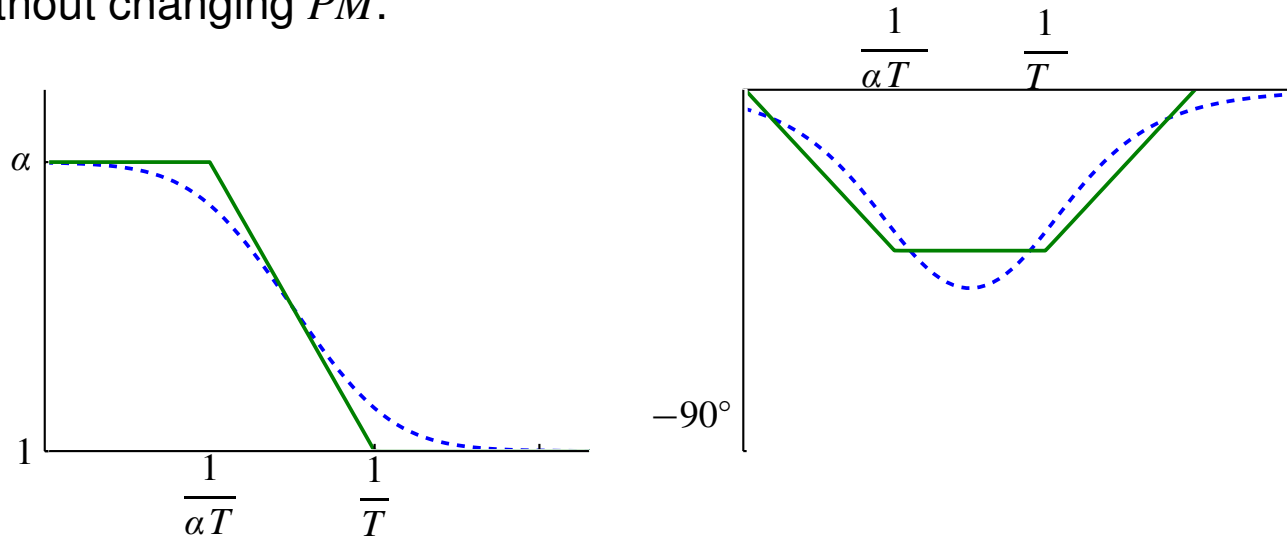
$$\frac{1}{T_I} \ll \omega_c.$$

### Lag compensation

- Approximates PI, but without integrator overflow.

$$D(s) = \alpha \left[ \frac{T s + 1}{\alpha T s + 1} \right] \quad \alpha > 1.$$

- Primary objective of lag is to add  $20 \log_{10} \alpha$  dB gain to *low* frequencies without changing *PM*.



- Steady-state response improves with little effect on transient response.

### Typical process

- Assumption is that we need to modify (increase) the dc-gain of the loop transfer function.
  - If we apply only a gain, then  $\omega_c$  typically increases and the phase margin decreases. *NOT GOOD*.
  - Instead, use lag compensation to lower high-frequency gain.
- Assume plant has gain  $K$  (adjustable), or that we insert a gain  $K$  into the system. Adjust the open-loop gain  $K$  to meet phase margin requirements (plus about  $5^\circ$  slop factor) at crossover without additional compensation.
- Draw Bode diagram of system using the gain  $K$  from above. Evaluate low-frequency gain.
- Determine  $\alpha$  to meet low-frequency gain requirements.

- Choose one corner frequency  $\omega = \frac{1}{T}$  (the zero) to be about one decade below crossover frequency. (This way, the phase added by the lag compensator will minimally affect  $PM$ . The phase added at crossover will be about  $5^\circ$ , hence our previous slop factor).
- The other corner frequency is at  $\omega = \frac{1}{\alpha T}$ .
- Iterate design to meet spec.

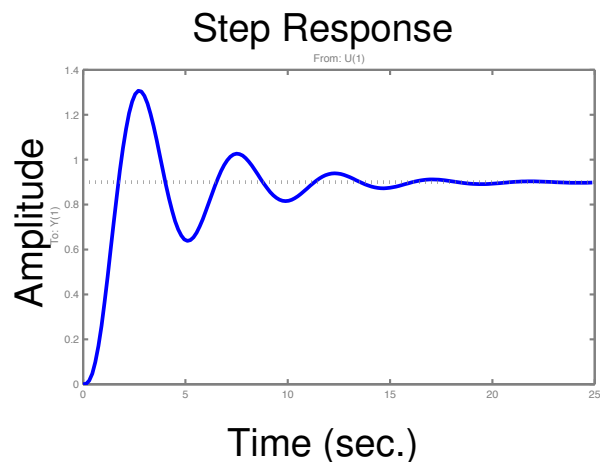
**EXAMPLE:**

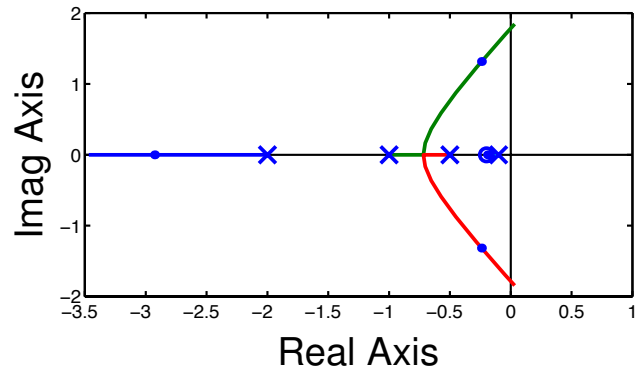
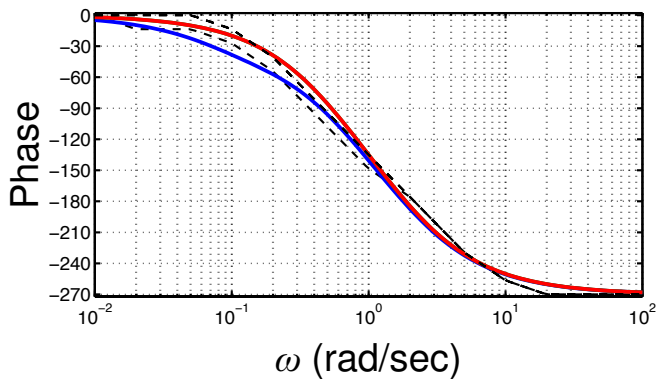
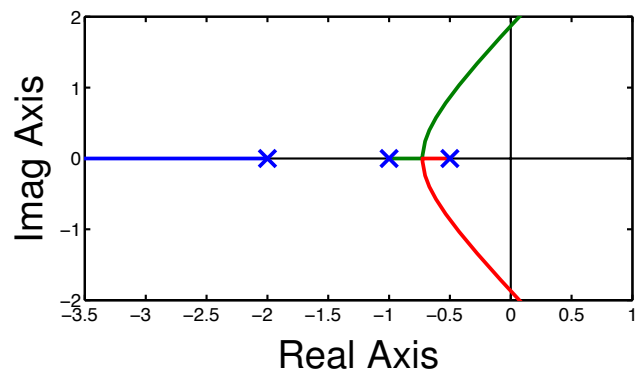
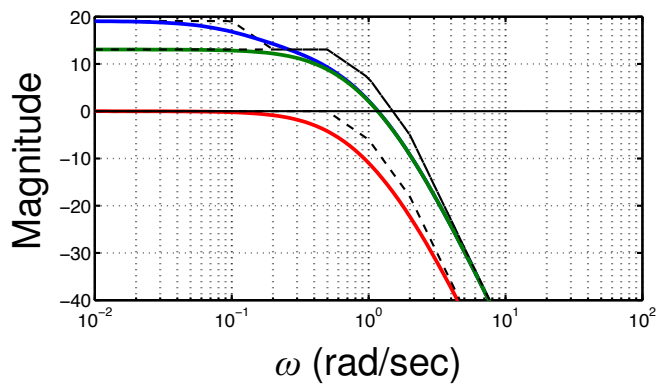
$$G(s) = \frac{K}{\left(\frac{1}{0.5}s + 1\right) (s + 1) \left(\frac{1}{2}s + 1\right)}.$$

- Bode plot for plant is red line, below. Uncompensated root locus is top-right plot.
- We want to design compensator for  $PM \geq 25^\circ$ ,  $K_p = 9$ .
  1. Set  $K = 4.5$  for  $PM = 30^\circ$ . This gives crossover at  $\omega_c \approx 1.2$  rads/sec. (Green line on Bode plot.)
  2. Low-frequency gain now about 13 dB or  $10^{(13/20)} = 4.5$ .
  3. Should be raised by a factor of 2 to get  $K_p = 9$ . So,  $\alpha = 2$ .
  4. Choose corner frequency at  $\omega = 0.2$  rads/sec.  $\frac{1}{T} = 0.2$ , or  $T = 5$ .
  5. We then know other corner frequency  $\omega = \frac{1}{\alpha T} = \frac{1}{10}$ .

- Altogether, we now have the compensator

$$D(s) = 2 \left[ \frac{5s + 1}{10s + 1} \right].$$



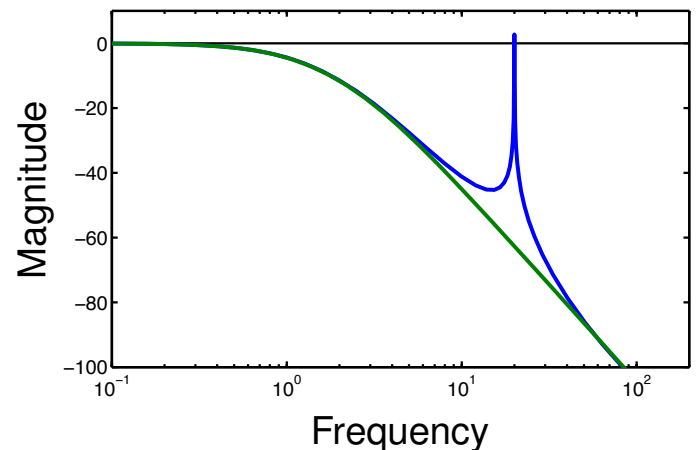


- Please understand that the “recipes” in these notes are not meant to be exhaustive!
- And, we often require both lead and lag to meet specs.
- Many other approaches to control design using frequency-response methods exist.
- In fact, once you are comfortable with Bode plots, you can add poles and zeros like LEGOs to build whatever frequency response you might like (within the limitations of Bode’s gain-phase theorem).
- Our next section gives some guidance on this.

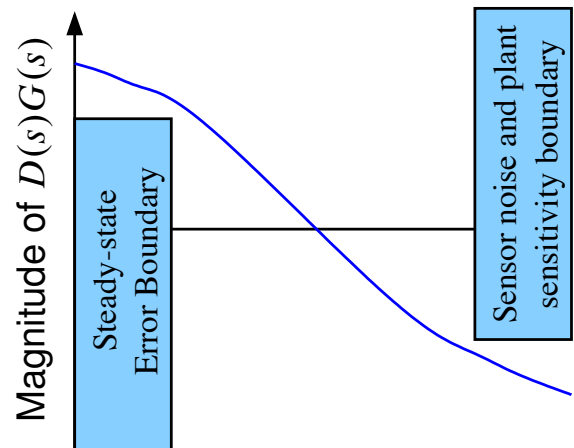
## 9.5: Design based on sensitivity

- Goal: Develop conditions on Bode plot of loop transfer function  $D(s)G(s)$  that will ensure good performance with respect to sensitivity, steady-state errors and sensor noise.
  - Steady-state performance = lower bound on system low-freq. gain.
  - Sensor noise = upper bound on high-frequency gain.

- Consider also the risk of an unmodeled system resonance:
- Magnitude may go over 1. Can cause instability. Must ensure that high-frequency gain is low so magnitude does not go over 1.

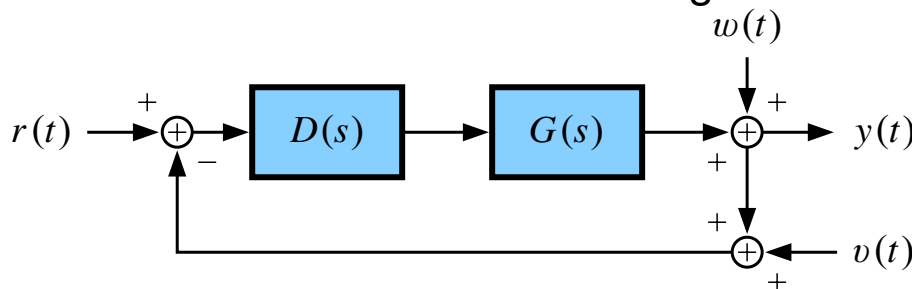


- Together, these help us design a desired loop frequency response:
  - Magnitude must be high at low frequencies, and
  - Low at high frequencies.



### Sensitivity functions

- The presence of noises also enters our design considerations.



$$Y(s) = W(s) + G(s)D(s)[R(s) - V(s) - Y(s)]$$

$$[1 + G(s)D(s)]Y(s) = W(s) + G(s)D(s)[R(s) - V(s)]$$

$$\text{or, } Y(s) = \frac{1}{1 + G(s)D(s)}W(s) + \frac{G(s)D(s)}{1 + G(s)D(s)}[R(s) - V(s)].$$

- Tracking error  $\triangleq R(s) - Y(s)$

$$\begin{aligned} E(s) &= R(s) - \frac{1}{1 + G(s)D(s)}W(s) - \frac{G(s)D(s)}{1 + G(s)D(s)}[R(s) - V(s)] \\ &= \frac{1}{1 + G(s)D(s)}[R(s) - W(s)] + \frac{1}{1 + G(s)D(s)}G(s)D(s)V(s) \end{aligned}$$

- Define the “sensitivity function”  $S(s)$  to be

$$S(s) \triangleq \frac{1}{1 + G(s)D(s)}$$

which is the transfer function from  $r(t)$  to  $e(t)$  and from  $w(t)$  to  $-e(t)$ .

- The “complementary sensitivity function”  $T(s) = 1 - S(s)$

$$1 - S(s) = \frac{G(s)D(s)}{1 + G(s)D(s)} = T(s)$$

which is the transfer function from  $r(t)$  to  $y(t)$ .

- If  $V = 0$ , then

$$Y(s) = S(s)W(s) + T(s)R(s)$$

and

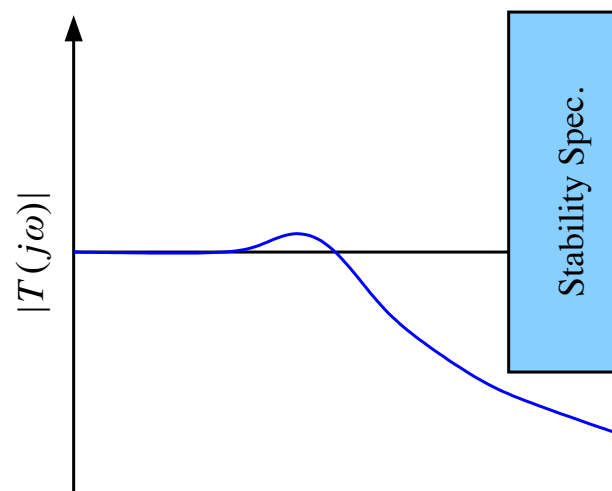
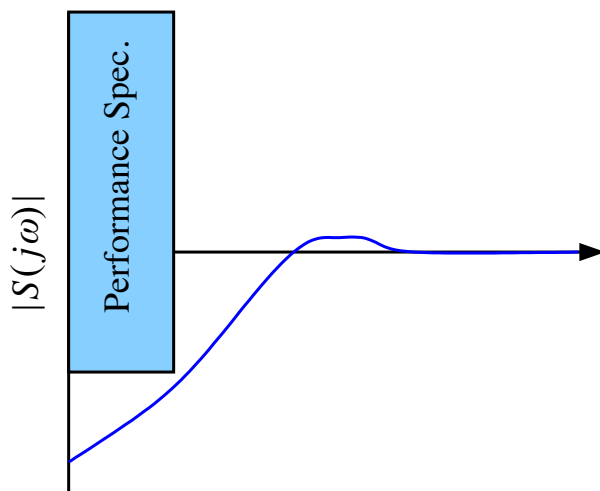
$$E(s) = S(s)[R(s) - W(s)].$$

- The sensitivity function here is related to the one we saw several weeks ago

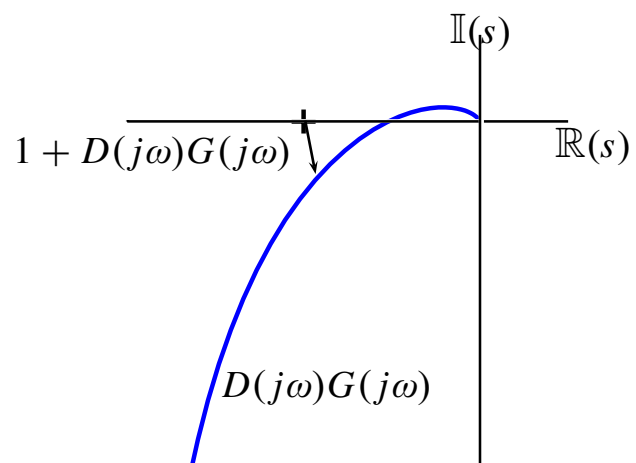
$$\begin{aligned} S_G^T &= \frac{\partial T}{\partial G} \cdot \frac{G}{T} = \frac{1 + D(s)G(s) - D(s)G(s)}{[1 + D(s)G(s)]^2} \cdot \frac{G(s)[1 + D(s)G(s)]}{G(s)} \\ &= \frac{1}{1 + D(s)G(s)} = S(s). \end{aligned}$$



- So  $S(s)$  is the sensitivity of the transfer function to plant perturbations.
- Recall that  $T(s) + S(s) = 1$ , regardless of  $D(s)$  and  $G(s)$ .
- We would like  $T(s) = 1$ . Then  $S(s) = 0$ ; Disturbance is cancelled, design is insensitive to plant perturbation, steady-state error  $\approx 0$ .
- *BUT*, for physical plants,  $G(s) \rightarrow 0$  for high frequencies (which forces  $S(s) \rightarrow 1$ ).
- Furthermore, the transfer function between  $V(s)$  and  $E(s)$  is  $T(s)$ . To reduce high frequency noise effects,  $T(s) \rightarrow 0$  as frequency increases, and  $S(s) \approx 1$ .
- Typical sensitivity and complementary sensitivity (closed-loop transfer) functions are:



- Another view of sensitivity:
- So,  $1 + D(j\omega)G(j\omega)$  is the distance between the Nyquist curve to the  $-1$  point, and
 
$$S(j\omega) = \frac{1}{1 + D(j\omega)G(j\omega)}.$$



- A large value of  $|S(j\omega)|$  indicates a nearly unstable Nyquist plot.
- The maximum value of  $|S|$  is a more accurate measure of stability than  $PM$  or  $GM$ . So, we want  $\max |S(j\omega)|$  small. How small?
- Note:  $E(j\omega) = S(j\omega)R(j\omega)$

$$|E(j\omega)| = |S(j\omega)R(j\omega)| \leq |S(j\omega)||R(j\omega)|$$

put a frequency-based error bound

$$|E(j\omega)| \leq |S(j\omega)||R(j\omega)| \leq e_b$$

- Let  $W_1(\omega) = R(j\omega)/e_b$ . Then,

$$|S(j\omega)| \leq \frac{1}{W_1(\omega)}$$

**EXAMPLE:** A unity-feedback system is to have an error less than 0.005 for all unity-amplitude sinusoids below 500 rads/sec. Draw  $|W_1(j\omega)|$  for this design.

- Spectrum of  $R(j\omega)$  is unity for  $0 \leq \omega \leq 500$ .
- Since  $e_b = 0.005$ ,

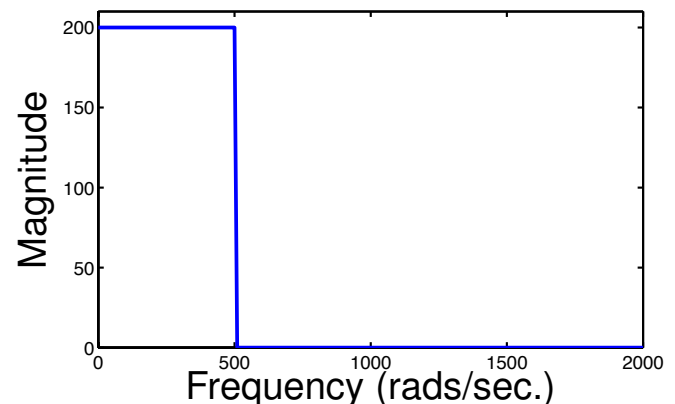
$$W_1(\omega) = \frac{1}{0.005} = 200$$

for this range.

- We can translate this requirement into a loop-gain requirement. When errors are small, loop gain is high, so  $|S(j\omega)| \approx \frac{1}{|D(j\omega)G(j\omega)|}$  and

$$\frac{1}{|D(j\omega)G(j\omega)|} \leq \frac{1}{W_1(\omega)}$$

or,  $|D(j\omega)G(j\omega)| \geq W_1(\omega)$

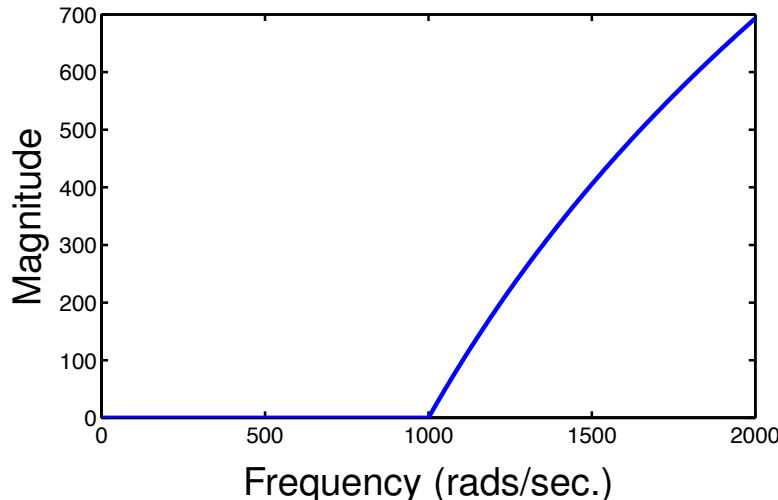


## 9.6: Robustness

- Typically there is some uncertainty in the plant transfer function. We want our design to be robustly stable, and to robustly give good performance (often called  $H_\infty$  design).
- Uncertainty often expressed as multiplicative

$$G(j\omega) = G_n(j\omega)[1 + W_2(j\omega)\Delta(j\omega)]$$

- $W_2(\omega)$  is a function of frequency expressing uncertainty, or size of possible error in transfer function as a function of frequency.
- $W_2(\omega)$  is almost always small at low frequencies.
- $W_2(\omega)$  increases at high frequencies as unmodeled structural flexibility is common.
- “Typical  $W_2$ ”



- $\Delta(j\omega)$  expresses uncertainty in phase. The only restriction is

$$|\Delta(j\omega)| \leq 1.$$

### Design

- Assume design for nominal plant  $G_n(s)$  is stable. Thus,  
 $1 + D(j\omega)G_n(j\omega) \neq 0 \forall \omega.$

- For robust stability,

$$1 + D(j\omega)G(j\omega) \neq 0 \forall \omega$$

$$1 + D(j\omega)G_n(j\omega)[1 + W_2(\omega)\Delta(j\omega)] \neq 0$$

$$\underbrace{\frac{1 + D(j\omega)G_n(j\omega)}{1 + D(j\omega)G_n(j\omega)}}_{\neq 0 \text{ by assumption}} + \frac{D(j\omega)G_n(j\omega)}{1 + D(j\omega)G_n(j\omega)}W_2(\omega)\Delta(j\omega) \neq 0$$

$$\text{recall, } T(j\omega) = \frac{D(j\omega)G_n(j\omega)}{1 + D(j\omega)G_n(j\omega)},$$

$$[1 + T(j\omega)W_2(\omega)\Delta(j\omega)] \neq 0$$

$$\text{so, } |T(j\omega)W_2(\omega)\Delta(j\omega)| < 1$$

$$\text{or, } |T(j\omega)W_2(\omega)| < 1.$$

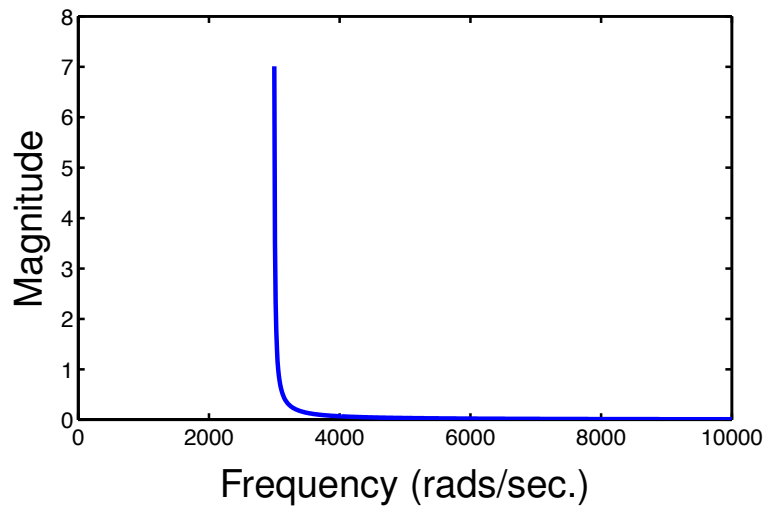
- For high frequencies  $D(j\omega)G_n(j\omega)$  is typically small, so  $T(j\omega) \approx D(j\omega)G_n(j\omega)$ . Thus

$$|D(j\omega)G_n(j\omega)W_2(\omega)| < 1$$

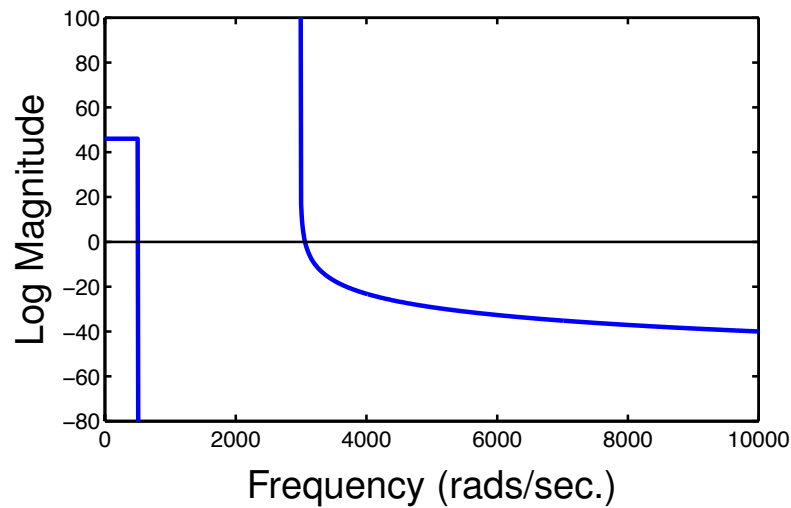
$$|D(j\omega)G_n(j\omega)| < \frac{1}{W_2(\omega)}.$$

**EXAMPLE:** The uncertainty in a plant model is described by a function  $W_2(\omega)$  which is zero until  $\omega = 3000$  rads/sec, and increases linearly from there to a value of 100 at  $\omega = 10,000$  rads/sec. It remains constant at 100 for higher frequencies. Plot constraint on  $D(j\omega)G_n(j\omega)$ .

- Where  $W_2(\omega) = 0$ , there is no constraint on the magnitude of the loop gain. Above  $\omega = 3000$ ,  $1/W_2(\omega)$  is a hyperbola from  $\infty$  to 0.01 at 10,000.



- Combining  $W_1(\omega)$  and  $W_2(\omega)$  requirements,



- Limitation: Crossover needs to be with slope  $\approx -1$ . So, cannot make constraints too strict, or design will be unstable.