## DYNAMIC RESPONSE

#### 3.1: System response in the time domain

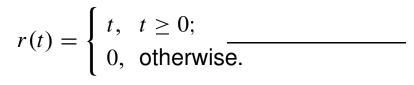
- We can now model dynamic systems with differential equations.
  What do these equations mean?
- We'll proceed by looking at a system's response to certain inputs in the time domain.
- Then, we'll see how the Laplace transform can make our lives a lot easier by simplifying the math.
- This will give insights into how we might specify the way the system should respond.
- Finally, we'll preview how adding dynamics (*e.g.*, a controller) can change how the system responds.

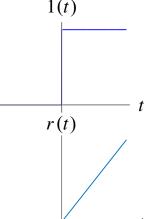
#### Some important input signals

- Several signals recur throughout this course.
- The unit step function:

$$1(t) = \begin{cases} 1, & t \ge 0; \\ 0, & \text{otherwise.} \end{cases}$$

■ The unit ramp function:

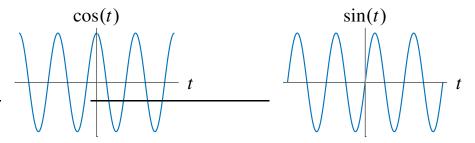




■ The unit parabola function:

$$p(t) = \begin{cases} \frac{t^2}{2}, & t \ge 0; \\ 0, & \text{otherwise.} \end{cases}$$

■ The cosine/sine functions:



- The (ideal) impulse function,  $\delta(t)$ :
  - Very strange "generalized" function, defined only under an integral.

$$\delta(t) = 0, \quad t \neq 0$$
 zero duration 
$$\int_{-\infty}^{\infty} \delta(t) \, \mathrm{d}t = 1.$$
 unit area.

Sifting property:<sup>1</sup>

$$\int_{-\infty}^{\infty} x(\tau)\delta(t-\tau) d\tau = x(t).$$

Symbol

<sup>&</sup>lt;sup>1</sup> Assumes that x(t) is continuous at  $t = \tau$ . Interpretation: no value of x(t) matters except that over the short range where  $\delta(t)$  occurs.

#### Time response of a linear time invariant system

■ Let y(t) be the output of an LTI system with input x(t).

$$y(t) = \mathbb{T}[x(t)]$$

$$= \mathbb{T}\left[\int_{-\infty}^{\infty} x(\tau)\delta(t-\tau) \,\mathrm{d}\tau\right] \qquad \text{(sifting)}$$

$$= \int_{-\infty}^{\infty} x(\tau)\mathbb{T}[\delta(t-\tau)] \,\mathrm{d}\tau. \qquad \text{(linear)}$$
Let  $h(t,\tau) = \mathbb{T}[\delta(t-\tau)]$ 

$$= \int_{-\infty}^{\infty} x(\tau)h(t,\tau) \,\mathrm{d}\tau$$

If the system is time invariant,  $h(t, \tau) = h(t - \tau)$ 

$$= \int_{-\infty}^{\infty} x(\tau)h(t-\tau) d\tau \qquad \text{(time invariant)}$$

$$\stackrel{\triangle}{=} x(t) * h(t).$$

- The output of an LTI system is equal to the <u>convolution</u> of its <u>impulse</u> <u>response</u> with the input.
- This makes life EASY (TRUST me!)

#### **EXAMPLE:** Finding an impulse response:

- Consider a first-order system,  $\dot{y}(t) + ky(t) = u(t)$ .
- Let  $y(0^-) = 0$ ,  $u(t) = \delta(t)$ .
- For positive time we have  $\dot{y}(t) + ky(t) = 0$ . Recall from your differential-equation math course:  $y(t) = Ae^{st}$ , solve for A, s.

$$\dot{y}(t) = Ase^{st}$$

$$Ase^{st} + kAe^{st} = 0$$

$$s + k = 0$$

$$s = -k$$

■ We have solved for s; now, solve for A.

$$\int_{0^{-}}^{0^{+}} \dot{y}(t) dt + k \int_{0^{-}}^{0^{+}} y(t) dt = \int_{0^{-}}^{0^{+}} \delta(t) dt$$

$$y(0^{+}) - y(0^{-}) = 1$$

$$Ae^{-k0^{+}} - 0 = 1$$

$$A = 1.$$

- Response to impulse:  $h(t) = e^{-kt}$ , t > 0.
- $\bullet h(t) = e^{-kt} 1(t).$
- Response of this system to general input:

$$y(t) = \int_{-\infty}^{\infty} h(\tau)u(t-\tau) d\tau$$
$$= \int_{-\infty}^{\infty} e^{-k\tau} 1(\tau)u(t-\tau) d\tau$$
$$= \int_{0}^{\infty} e^{-k\tau} u(t-\tau) d\tau.$$

#### 3.2: Transfer functions

- Response to impulse = "impulse response": h(t).
- Response to general input = messy convolution: h(t) \* u(t).
- To choose a simpler example, what is the response to a cosine?

$$A\cos(\omega t) = \frac{A}{2} \left( e^{j\omega t} + e^{-j\omega t} \right)$$

#### Break it down: What is the response to an exponential?

■ Let  $u(t) = e^{st}$ , where s is complex.

$$y(t) = \int_{-\infty}^{\infty} h(\tau)u(t-\tau) d\tau = \int_{-\infty}^{\infty} h(\tau)e^{s(t-\tau)} d\tau$$

$$= \int_{-\infty}^{\infty} h(\tau)e^{st}e^{-s\tau} d\tau$$

$$= e^{st} \int_{-\infty}^{\infty} h(\tau)e^{-s\tau} d\tau$$
Transfer function,  $H(s)$ 

$$= e^{st} H(s).$$

■ An  $e^{st}$  input decouples the convolution into two *independent* parts: a part depending on  $e^{st}$  and a part depending on h(t).

**EXAMPLE:** 
$$\dot{y}(t) + ky(t) = u(t) = e^{st}$$
:

but,  $y(t) = H(s)e^{st}$ ,  $\dot{y}(t) = sH(s)e^{st}$ ,

 $sH(s)e^{st} + kH(s)e^{st} = e^{st}$ 
 $H(s) = \frac{1}{s+k}$  (I never integrated!)

 $y(t) = \frac{e^{st}}{s+k}$ .

### Response to a cosinusoid (revisited)

Let 
$$s=j\omega$$
  $u(t)=e^{j\omega t}$   $y(t)=H(j\omega)e^{j\omega t}$   $s=-j\omega$   $u(t)=e^{-j\omega t}$   $y(t)=H(-j\omega)e^{-j\omega t}$   $y(t)=A\cos(\omega t)$   $y(t)=\frac{A}{2}\left[H(j\omega)e^{j\omega t}+H(-j\omega)e^{-j\omega t}\right]$ 

Now, 
$$H(j\omega) \stackrel{\triangle}{=} Me^{j\phi}$$
  $H(-j\omega) = Me^{-j\phi}$  (can be shown for  $h(t)$  real)  $y(t) = \frac{AM}{2} \left[ e^{j(\omega t + \phi)} + e^{-j(\omega t + \phi)} \right]$   $= AM \cos(\omega t + \phi).$ 

■ The response of an LTI system to a sinusoid is a sinusoid! (of the same frequency).

**EXAMPLE:** Frequency response of our first order system:

$$H(s) = \frac{1}{s+k}$$

$$H(j\omega) = \frac{1}{j\omega + k}$$

$$M = |H(j\omega)| = \frac{1}{\sqrt{\omega^2 + k^2}}$$

$$\phi = \angle H(j\omega) = -\tan^{-1}\left(\frac{\omega}{k}\right)$$

$$y(t) = \frac{A}{\sqrt{\omega^2 + k^2}}\cos\left(\omega t - \tan^{-1}\left(\frac{\omega}{k}\right)\right).$$

Can we use these results to simplify convolution and get an easier way to understand dynamic response?

#### **Defining the Laplace** $\mathcal{L}_{-}$ **transform**

■ We have seen that if a system has an impulse response h(t), we can compute a transfer function H(s),

$$H(s) = \int_{-\infty}^{\infty} h(t)e^{-st} dt.$$

■ Since we deal with causal systems (possibly with an impulse at t = 0), we can integrate from  $0^-$  instead of negative infinity.

$$H(s) = \int_{0^{-}}^{\infty} h(t)e^{-st} dt.$$

■ This is called the one-sided (uni-lateral) Laplace transform of h(t).

#### **Laplace Transforms of Common Signals**

Name	Time function, $f(t)$	Laplace $tx., F(s)$
Unit impulse	$\delta(t)$	1
Unit step	1( <i>t</i> )	1 s 1
Unit ramp	$t \cdot 1(t)$	$\frac{1}{s^2}$
nth order ramp	$t^n \cdot 1(t)$	$\frac{n!}{s^{n+1}}$
Exponential	$\exp(-at)1(t)$	$\frac{1}{s+a}$
Ramped exponential	$t\exp(-at)1(t)$	$\frac{1}{(s+a)^2}$
Sine	$\sin(bt)1(t)$	$\frac{b}{s^2+b^2}$
Cosine	$\cos(bt)1(t)$	$\frac{s}{s^2+b^2}$
Damped sine	$e^{-at}\sin(bt)1(t)$	$\frac{b}{(s+a)^2+b^2}$
Damped cosine	$e^{-at}\cos(bt)1(t)$	
Diverging sine	$t\sin(bt)1(t)$	$\frac{(s+a)^2 + b^2}{2bs}$ $\frac{(s^2 + b^2)^2}{(s^2 + b^2)^2}$
Diverging cosine	$t\cos(bt)1(t)$	$\frac{(s^2 + b^2)^2}{s^2 - b^2}$ $\frac{(s^2 + b^2)^2}{(s^2 + b^2)^2}$

#### **Properties of the Laplace transform**

- Superposition:  $\mathcal{L}\left\{af_1(t) + bf_2(t)\right\} = aF_1(s) + bF_2(s)$ .
- Time delay:  $\mathcal{L}\{f(t-\tau)\}=e^{-s\tau}F(s)$ .
- Time scaling:  $\mathcal{L}\{f(at)\} = \frac{1}{|a|}F\left(\frac{s}{a}\right)$ . (useful if original equations are expressed poorly in time scale. *e.g.*, measuring disk-drive seek speed in hours).
- Differentiation:

$$\mathcal{L}\left\{\dot{f}(t)\right\} = sF(s) - f(0^{-})$$

$$\mathcal{L}\left\{\dot{f}(t)\right\} = s^{2}F(s) - sf(0^{-}) - \dot{f}(0^{-})$$

$$\mathcal{L}\left\{f^{(m)}(t)\right\} = s^{m}F(s) - s^{m-1}f(0^{-}) - \dots - f^{(m-1)}(0^{-}).$$

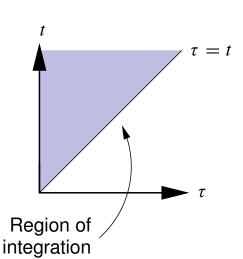
- Integration:  $\mathcal{L}\left\{\int_{0^{-}}^{t} f(\tau) d\tau\right\} = \frac{1}{s}F(s).$
- Convolution: Recall that y(t) = h(t) \* u(t)

$$Y(s) = \mathcal{L} \{y(t)\} = \mathcal{L} \{h(t) * u(t)\}$$

$$= \mathcal{L} \left\{ \int_{\tau=0^{-}}^{t} h(\tau)u(t-\tau) d\tau \right\}$$

$$= \int_{t=0^{-}}^{\infty} \int_{\tau=0^{-}}^{t} h(\tau)u(t-\tau) d\tau e^{-st} dt$$

$$= \int_{\tau=0^{-}}^{\infty} \int_{t=\tau^{-}}^{\infty} h(\tau)u(t-\tau) e^{-st} dt d\tau.$$



■ Multiply by  $e^{-s\tau}e^{s\tau}$ 

$$Y(s) = \int_{\tau=0^-}^{\infty} h(\tau)e^{-s\tau} \int_{t=\tau^-}^{\infty} u(t-\tau)e^{-s(t-\tau)} dt d\tau.$$

Let  $t' = t - \tau$ :

$$Y(s) = \int_{\tau=0^{-}}^{\infty} h(\tau)e^{-s\tau} d\tau \int_{t'=0^{-}}^{\infty} u(t')e^{-st'} dt'$$
  
$$Y(s) = H(s)U(s).$$

- The Laplace transform "unwraps" convolution for *general* input signals. Makes system easy to analyze.
- This is **the most** important property of the Laplace transform. This is why we use it. It converts differential equations into algebraic equations that we can solve quite readily.

Assume:

#### 3.3: The inverse Laplace transform

- The inverse Laplace transform converts  $F(s) \rightarrow f(t)$ .
- Once we get an intuitive feel for F(s), we won't need to do this often.
- The main tool for ILT is partial-fraction-expansion.

Assume: 
$$F(s) = \frac{b_0 s^m + b_1 s^{m-1} + \dots + b_m}{s^n + a_1 s^{n-1} + \dots + a_n}$$

$$= k \frac{\prod_{i=1}^m (s - z_i)}{\prod_{i=1}^n (s - p_i)} \qquad \text{(zeros)} \qquad \text{(poles)}$$

$$= \frac{c_1}{s - p_1} + \frac{c_2}{s - p_2} + \dots + \frac{c_n}{s - p_n} \qquad \text{if } \{p_i\} \text{ distinct.}$$
so, 
$$(s - p_1) F(s) = c_1 + \frac{c_2 (s - p_1)}{s - p_2} + \dots + \frac{c_n (s - p_1)}{s - p_n}$$

$$|\text{let } s = p_1: \qquad c_1 = (s - p_1) F(s)|_{s = p_1}$$

$$c_i = (s - p_i) F(s)|_{s = p_i}$$

$$f(t) = \sum_{i=1}^n c_i e^{p_i t} 1(t) \qquad \text{since } \mathcal{L}\left[e^{kt} 1(t)\right] = \frac{1}{s - k}.$$

$$\text{EXAMPLE: } F(s) = \frac{5}{s^2 + 3s + 2} = \frac{5}{(s + 1)(s + 2)}.$$

$$c_1 = (s + 1) F(s) \Big|_{s = -1} = \frac{5}{s + 2} \Big|_{s = -1} = 5$$

$$c_2 = (s + 2) F(s) \Big|_{s = -2} = \frac{5}{s + 1} \Big|_{s = -2} = -5$$

$$f(t) = (5e^{-t} - 5e^{-2t}) 1(t).$$

• If F(s) has repeated roots, we must modify the procedure. e.g., repeated three times:

$$F(s) = \frac{k}{(s - p_1)^3 (s - p_2) \cdots}$$

$$= \frac{c_{1,1}}{s - p_1} + \frac{c_{1,2}}{(s - p_1)^2} + \frac{c_{1,3}}{(s - p_1)^3} + \frac{c_2}{s - p_2} + \cdots$$

$$c_{1,3} = (s - p_1)^3 F(s) \Big|_{s = p_1}$$

$$c_{1,2} = \left[ \frac{d}{ds} \left( (s - p_1)^3 F(s) \right) \Big|_{s = p_1}$$

$$c_{1,1} = \frac{1}{2} \left[ \frac{d^2}{ds^2} \left( (s - p_1)^3 F(s) \right) \Big|_{s = p_1}$$

$$c_{x,k-i} = \frac{1}{i!} \left[ \frac{d^i}{ds^i} \left( (s - p_i)^k F(s) \right) \Big|_{s = p_i}$$

**EXAMPLE:** Find the ILT of

$$H(s) = \frac{s+2}{(s+1)^2(s+3)} = \frac{A}{s+1} + \frac{B}{(s+1)^2} + \frac{C}{s+3}.$$

• We start with B,

$$B = \frac{s+2}{s+3} \bigg|_{s=-1} = \frac{1}{2}.$$

 $\blacksquare$  Next, we find A,

$$A = \left[ \frac{d}{ds} \left( \frac{s+2}{s+3} \right) \right|_{s=-1}$$

$$= \left[ \frac{d}{ds} (s+2)(s+3)^{-1} \right|_{s=-1}$$

$$= \left[ (s+2)(-1)(s+3)^{-2} + (s+3)^{-1} \right|_{s=-1}$$

$$= \left[ -\frac{s+2}{(s+3)^2} + \frac{1}{s+3} \right|_{s=-1}$$

$$= -\frac{1}{4} + \frac{1}{2} = \frac{1}{4}.$$

 $\blacksquare$  Lastly, we find C,

$$C = \frac{s+2}{(s+1)^2} \bigg|_{s=-3} = -\frac{1}{4}.$$

■ Therefore, the inverse Laplace transform we are looking for is

$$h(t) = \left[ \frac{1}{2} t e^{-t} + \frac{1}{4} e^{-t} - \frac{1}{4} e^{-3t} \right] 1(t).$$

**EXAMPLE:** Find ILT of  $\frac{s+3}{(s+1)(s+2)^2}$ .

■ ans:  $f(t) = (2e^{-t} - 2e^{-2t} - te^{-2t})1(t)$ . from repeated root.

- Note that this is quite tedious, but MATLAB can help.
- Try MATLAB with two examples; first,  $F(s) = \frac{5}{s^2 + 3s + 2}$ .

#### Example 1.

# 

#### Example 2.

```
>> Fnum = [0 0 1 3];
>> Fden = conv([1 1],conv([1 2],[1 2]));
[r,p,k] = residue(Fnum,Fden);
r = -2
     -1
     2
p = -2
     -2
     -1
k = []
```

■ When you use "residue" and get repeated roots, *BE SURE* to type "help residue" to correctly interpret the result.

#### **Using the Laplace transform to solve problems**

We can use the Laplace transform to solve both homogeneous and forced differential equations.

**EXAMPLE:** 
$$\ddot{y}(t) + y(t) = 0$$
,  $y(0^{-}) = \alpha$ ,  $\dot{y}(0^{-}) = \beta$ .

■ Take Laplace transforms, term by term:

$$s^{2}Y(s) - \alpha s - \beta + Y(s) = 0$$

$$Y(s)(s^{2} + 1) = \alpha s + \beta$$

$$Y(s) = \frac{\alpha s + \beta}{s^{2} + 1}$$

$$= \frac{\alpha s}{s^{2} + 1} + \frac{\beta}{s^{2} + 1}$$

$$\frac{\alpha}{s^{2} + a^{2}} \iff \cos(at)1(t)$$

$$\frac{s}{s^{2} + a^{2}} \iff \cos(at)1(t)$$

- From tables,  $y(t) = [\alpha \cos(t) + \beta \sin(t)]1(t)$ .
- If initial conditions are zero, things are very simple.

#### **EXAMPLE:**

$$\ddot{y}(t) + 5\dot{y}(t) + 4y(t) = u(t),$$
  $y(0^{-}) = 0, \ \dot{y}(0^{-}) = 0, \ u(t) = 2e^{-2t}1(t).$ 

Start with:

$$s^{2}Y(s) + 5sY(s) + 4Y(s) = \frac{2}{s+2}$$

■ Rearrange:

$$Y(s) = \frac{2}{(s+2)(s+1)(s+4)}$$
$$= \frac{-1}{s+2} + \frac{2/3}{s+1} + \frac{1/3}{s+4}.$$

■ From tables, 
$$y(t) = \left[ -e^{-2t} + \frac{2}{3}e^{-t} + \frac{1}{3}e^{-4t} \right] 1(t)$$
.

$$H(s) \iff h(t)$$

$$H(s+a) \iff e^{-at}h(t).$$

#### 3.4: Time response versus pole locations

- If we wish to know how a system responds to some input (for example, an impulse response, or a step response), it seems like we need to do the following:
  - 1. Find the Laplace transform U(s) of the input u(t),
  - 2. Find the Laplace transform of the output Y(s) = H(s)U(s),
  - 3. Find the time response by taking the inverse Laplace transform of Y(s). That is,  $y(t) = \mathcal{L}^{-1}(Y(s))$ .
- This is true if we want a precise, *quantitative* answer.
- But, if we're interested only in a *qualitative* answer, we can learn a lot simply by looking at the pole locations of the transfer function.
- If we can represent  $H(s) = \text{num}_H(s)/\text{den}_H(s)$  and  $U(s) = \text{num}_U(s)/\text{den}_U(s)$ , then we have

$$Y(s) = \frac{\text{num}_{H}(s)\text{num}_{U}(s)}{\text{den}_{H}(s)\text{den}_{U}(s)}$$
$$= \sum_{k} \frac{r_{k}}{s + p_{k}},$$

where "pole"  $s = -p_k$  is a root of either  $den_H(s)$  or  $den_U(s)$ .

- So, some of the system's response is due to the poles of the input signal, and some is due to the poles of the plant.
- Here, we're interested in the contribution due to the poles of the plant.
  - Neglecting the residues  $r_k$ , which simply scale the output by some fixed amount, we're interested in "what does an output of the type  $\frac{1}{s+p_k}$  look like?"

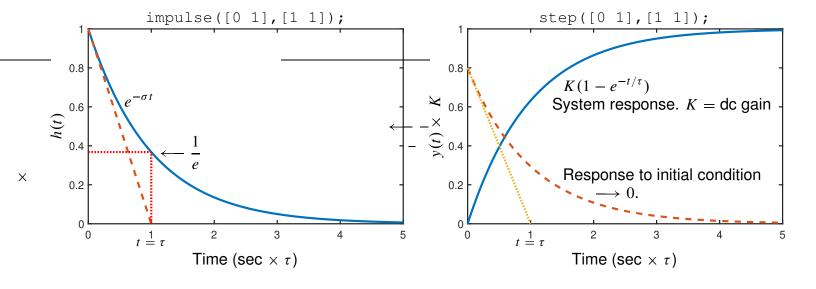
- That is, the poles *qualitatively* determine the behavior of the system; zeros (equivalently, residues) quantify this relationship.
- Note that the poles  $p_k$  may be real, or they may occur in complex-conjugate pairs.
- So, in the next sections, we look at the time responses of real poles and of complex-conjugate poles.

#### Time response due to a real pole

Consider a transfer function having a single real pole:

$$H(s) = \frac{1}{s + \sigma} \quad \Longrightarrow \quad h(t) = e^{-\sigma t} 1(t).$$

- If  $\sigma > 0$ , pole is at s < 0, STABLE i.e., impulse response decays, and any bounded input produces bounded output.
- If  $\sigma$  < 0, pole is at s > 0, *UNSTABLE*.
- $\sigma$  is "time constant" factor:  $\tau = 1/\sigma$ .



#### Time response due to complex-conjugate poles

 Now, consider a <u>second-order</u> transfer function having complex-conjugate poles

$$H(s) = \frac{b_0}{s^2 + a_1 s + a_2} = K \underbrace{\frac{\omega_n^2}{s^2 + 2\zeta \omega_n s + \omega_n^2}}_{\text{standard form}}.$$

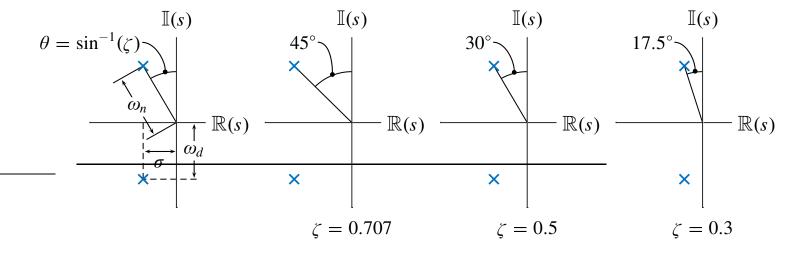
 $\zeta =$  damping ratio.

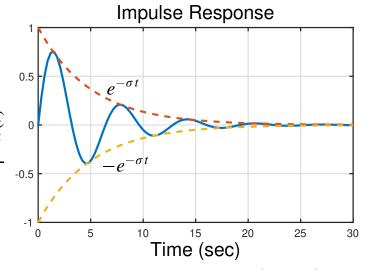
 $\omega_n$  = natural frequency or undamped frequency.

$$h(t) = \frac{\omega_n}{\sqrt{1 - \zeta^2}} e^{-\sigma t} \left( \sin(\omega_d t) \right) 1(t),$$

where,  $\sigma = \zeta \omega_n$ ,

$$\omega_d = \omega_n \sqrt{1 - \zeta^2} = \text{damped frequency}.$$



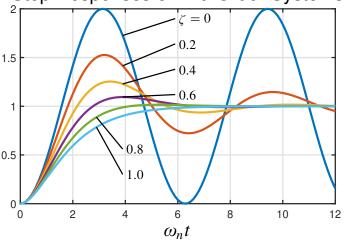


Envelope of sinusoid decays as  $e^{-\sigma t}$ 

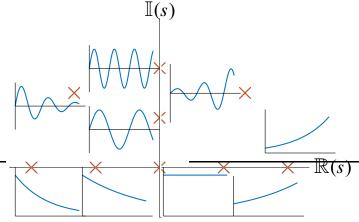
Impulse Responses of 2nd-Order Systems

 $\zeta = 0$  0.5 0  $\zeta = 1$  0.8 0.6 0 0.6 0 0.6 0 0.7 0.8 0.9 0.8 0.9 0.8 0.9 0.

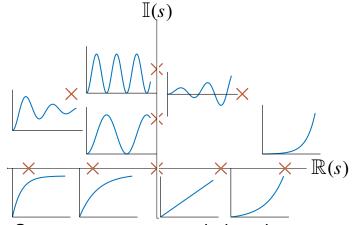
Step Responses of 2nd-Order Systems



■ Low damping,  $\zeta \approx 0$ , oscillatory; High damping,  $\zeta \approx 1$ , no oscillations.



Impulse responses vs. pole locations

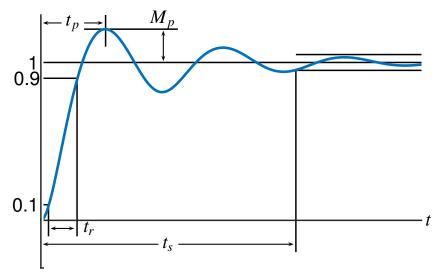


Step responses vs. pole locations

- $0 < \zeta < 1$  underdamped.
- $\zeta = 1$  critically damped,  $\zeta > 1$  over-damped.

#### 3.5: Time-domain specifications

- We have seen impulse and step responses for first- and second-order systems.
- Our control problem may be to specify exactly what the response SHOULD be.
- Usually expressed in terms of the step response.



- $t_r$  = Rise time = time to reach vicinity of new set point.
- $t_s$  = Settling time = time for transients to decay (to 5 %, 2 %, 1 %).
- $M_p$  = Percent overshoot.
- $t_p$  = Time to peak.

#### Rise Time

- All step responses rise in roughly the same amount of time (see pg. 3–17.) Take  $\zeta = 0.5$  to be average.
  - time from 0.1 to 0.9 is approximately  $\omega_n t_r = 1.8$ :

$$t_r \approx \frac{1.8}{\omega_n}$$

- We could make this more accurate, but note:
  - Only valid for 2nd-order systems with no zeros.
  - Use this as approximate design "rule of thumb" and iterate design until spec. is met.

#### Peak Time and Overshoot

■ Step response can be found from ILT of H(s)/s.

$$y(t) = 1 - e^{-\sigma t} \left( \cos(\omega_d t) + \frac{\sigma}{\omega_d} \sin(\omega_d t) \right),$$

$$\omega_d = \omega_n \sqrt{1 - \zeta^2}, \quad \sigma = \zeta \omega_n.$$

■ Peak occurs when  $\dot{y}(t) = 0$ 

$$\dot{y}(t) = \sigma e^{-\sigma t} \left( \cos(\omega_d t) + \frac{\sigma}{\omega_d} \sin(\omega_d t) \right) - e^{-\sigma t} \left( -\omega_d \sin(\omega_d t) + \sigma \cos(\omega_d t) \right)$$
$$= e^{-\sigma t} \left( \frac{\sigma^2}{\omega_d} \sin(\omega_d t) + \omega_d \sin(\omega_d t) \right) = 0.$$

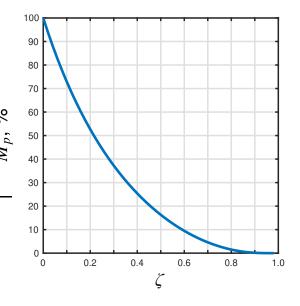
■ So,

$$\omega_d t_p = \pi,$$

$$t_p = \frac{\pi}{\omega_d} = \frac{\pi}{\omega_n \sqrt{1 - \zeta^2}}.$$

 $\blacksquare M_p = e^{-\zeta \pi/\sqrt{1-\zeta^2}} \times 100.$ 

• (common values:  $M_p = 16\%$  for  $\zeta = 0.5$ ;  $M_p = 5\%$  for  $\zeta = 0.7$ ).



#### Settling Time

Determined mostly by decaying exponential

$$e^{-\omega_n \zeta t_s} = \epsilon$$
 ...  $\epsilon = 0.01, 0.02, \text{ or } 0.05$ 

#### **EXAMPLE:**

$$\epsilon = 0.01$$

$$e^{-\omega_n \zeta t_s} = 0.01$$

$$\omega_n \zeta t_s = 4.6$$

$$t_s = \frac{4.6}{\zeta \omega_n} = \frac{4.6}{\sigma}$$

$$\epsilon \qquad t_s = \frac{4.6}{\sigma}$$

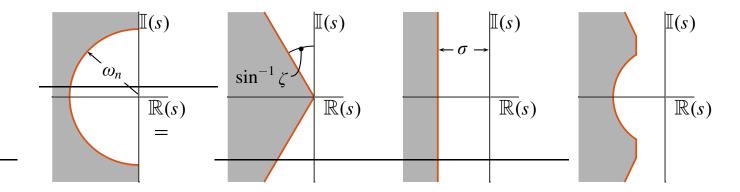
$$0.01 \quad t_s = 4.6/\sigma$$

$$0.02 \quad t_s = 3.9/\sigma$$

$$0.05 \quad t_s = 3.0/\sigma$$

### **Design synthesis**

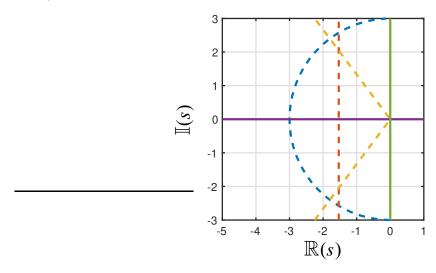
- Specifications on  $t_r$ ,  $t_s$ ,  $M_p$  determine pole locations.
- $\bullet \omega_n \geq 1.8/t_r$ .
- $\zeta \ge \text{fn}(M_p)$ . (read off of  $\zeta$  versus  $M_p$  graph on page 3–19)
- $\sigma \ge 4.6/t_s$ . (for example—settling to 1%)



**EXAMPLE:** Converting specs. to *s*-plane

■ Specs:  $t_r \le 0.6$ ,  $M_p \le 10\%$ ,  $t_s \le 3$  sec. at 1%

- $\omega_n \ge 1.8/t_r = 3.0 \text{ rad/sec.}$
- From graph of  $M_p$  versus  $\zeta$ ,  $\zeta \geq 0.6$ .
- $\sigma \ge 4.6/3 = 1.5$  sec.



#### **EXAMPLE:** Designing motor compensator

Suppose a servo-motor system for a pen-plotter has transfer function

$$\frac{0.5K_a}{s^2 + 2s + 0.5K_a} = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}.$$

- Only one adjustable parameter  $K_a$ , so can choose only one spec:  $t_r$ ,  $t_s$  or  $M_p$   $\Longrightarrow$  Allow NO overshoot.
- $M_p = 0, \ \zeta = 1.$
- From transfer function:  $2 = 2\zeta \omega_n$   $\omega_n = 1$ .
- $\bullet \omega_n^2 = 1^2 = 0.5 K_a, \quad K_a = 2.0$
- Note:  $t_s = 4.6$  seconds. We will need a better controller than this for a pen plotter!

### 3.6: Time response vs. pole locations: Higher order systems

■ We have looked at first-order and second-order systems without zeros, and with unity gain.

#### Non-unity gain

■ If we multiply by K, the dc gain is K.  $t_r$ ,  $t_s$ ,  $M_p$ ,  $t_p$  are not affected.

#### Add a zero to a second-order system

$$H_1(s) = \frac{2}{(s+1)(s+2)}$$

$$= \frac{2}{s+1} - \frac{2}{s+2}$$

$$H_2(s) = \frac{2(s+1.1)}{1.1(s+1)(s+2)}$$

$$= \frac{2}{1.1} \left( \frac{0.1}{s+1} + \frac{0.9}{s+2} \right)$$

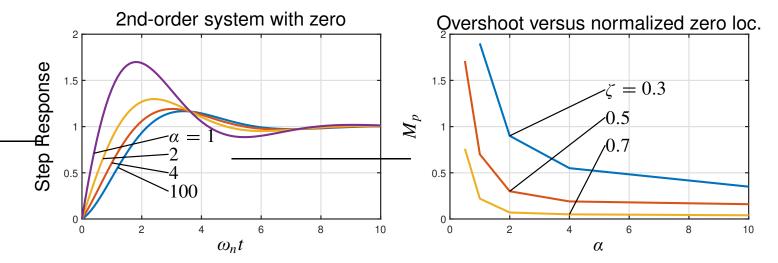
$$= \frac{0.18}{s+1} + \frac{1.64}{s+2}$$

- Same dc gain (at s = 0).
- Coefficient of (s + 1) pole *GREATLY* reduced.
- General conclusion: a zero "near" a pole tends to cancel the effect of that pole.
- How about transient response?

$$H(s) = \frac{\frac{s}{\alpha \zeta \omega_n} + 1}{(s/\omega_n)^2 + 2\zeta s/\omega_n + 1}.$$

- Zero at  $s = -\alpha \sigma$  (since  $\sigma = \zeta \omega_n$ ).
- Poles at  $\mathbb{R}(s) = -\sigma$ .
- Large  $\alpha$ , zero far from poles  $\Longrightarrow$  no effect.
- $\blacksquare \alpha \approx 1$ , large effect.

■ Notice that the overshoot goes up as  $\alpha \to 0$ .



■ A little more analysis; set  $\omega_n = 1$ 

$$H(s) = \frac{\frac{s}{\alpha \zeta} + 1}{s^2 + 2\zeta s + 1}$$

$$= \frac{1}{s^2 + 2\zeta s + 1} + \left(\frac{1}{\alpha \zeta}\right) \frac{s}{s^2 + 2\zeta s + 1}$$

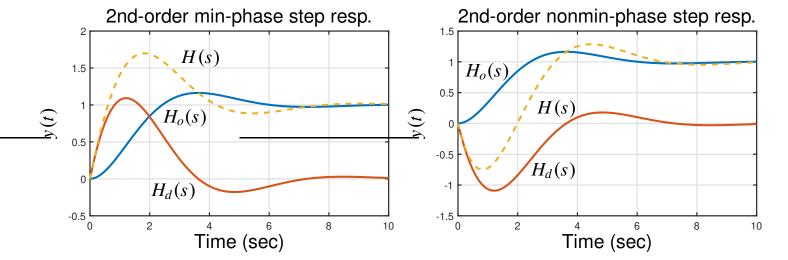
$$= H_o(s) + H_d(s).$$

- $H_o(s)$  is the original response, without the zero.
- $H_d(s)$  is the added term due to the zero. Notice that

$$H_d(s) = \frac{1}{\alpha \zeta} s H_o(s).$$

The time response is a scaled version of the *derivative* of the time response of  $H_o(s)$ .

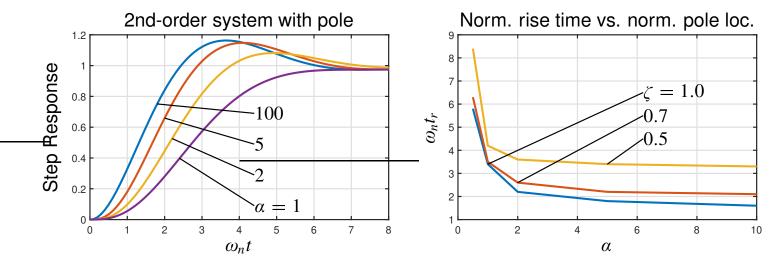
■ If any of the zeros in RHP ( $\alpha < 0$ ), system is nonminimum phase.



#### Add a pole to a second order system

$$H(s) = \frac{1}{\left(\frac{s}{\alpha \zeta \omega_n} + 1\right) \left[ (s/\omega_n)^2 + 2\zeta s/\omega_n + 1 \right]}.$$

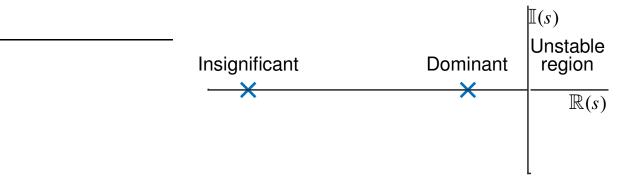
- Original poles at  $\mathbb{R}(s) = -\sigma = -\zeta \omega_n$ .
- New pole at  $s = -\alpha \zeta \omega_n$ .
- Major effect is an increase in rise time.



### **Summary of higher-order approximations**

■ Extra zero in LHP will increase overshoot if the zero is within a factor of  $\approx$  4 from the real part of complex poles.

- Extra zero in RHP depresses overshoot, and may cause step response to start in wrong direction. *DELAY*.
- Extra pole in LHP increases rise-time if extra pole is within a factor of  $\approx$  4 from the real part of complex poles.



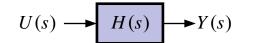
- MATLAB 'step' and 'impulse' commands can plot higher order system responses.
- Since a model is an approximation of a true system, it may be all right to reduce the order of the system to a first or second order system. If higher order poles and zeros are a factor of 5 or 10 time farther from the imaginary axis.
  - Analysis and design much easier.
  - Numerical accuracy of simulations better for low-order models.
  - 1st- and 2nd-order models provide us with great intuition into how the system works.
  - May be just as accurate as high-order model, since high-order model itself may be inaccurate.

#### 3.7: Changing dynamic response

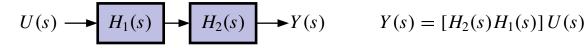
- Topic of the rest of the course.
- Important tool: block diagram manipulation.

#### Block-diagram manipulation

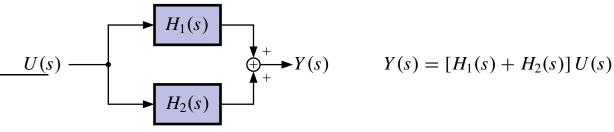
- We have already seen block diagrams (see pg. 1–4).
- Shows information/energy flow in a system, and when used with Laplace transforms, can simplify complex system dynamics.
- Four *BASIC* configurations:



$$Y(s) = H(s)U(s)$$



$$Y(s) = [H_2(s)H_1(s)] U(s)$$



$$Y(s) = [H_1(s) + H_2(s)] U(s)$$

$$R(s) \xrightarrow{+U_1(s)} H_1(s) \xrightarrow{+V_1(s)} Y(s)$$

$$Y_2(s) \xrightarrow{+U_1(s)} H_2(s) \xrightarrow{+V_1(s)} U_2(s)$$

$$U_{1}(s) = R(s) - Y_{2}(s)$$

$$Y_{2}(s) = H_{2}(s)H_{1}(s)U_{1}(s)$$

$$SO, U_{1}(s) = R(s) - H_{2}(s)H_{1}(s)U_{1}(s)$$

$$= \frac{R(s)}{1 + H_{2}(s)H_{1}(s)}$$

$$Y(s) = H_{1}(s)U_{1}(s)$$

$$= \frac{H_{1}(s)}{1 + H_{2}(s)H_{1}(s)}R(s)$$

#### Alternate representation

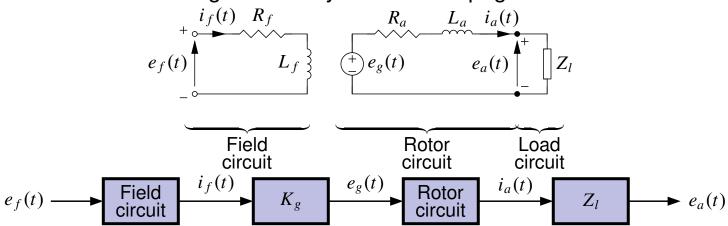
$$U(s) \xrightarrow{H_1(s)} Y(s)$$

$$U(s) \xrightarrow{H_1(s)} H_2(s) \xrightarrow{Y(s)} Y(s)$$

$$U(s) \xrightarrow{H_2(s)} Y(s)$$

$$R(s) \xrightarrow{H_1(s)} Y(s)$$

### **EXAMPLE:** Recall dc generator dynamics from page 2–19



■ Compute the transfer functions of the four blocks.

$$e_f(t) = R_f i_f(t) + L_f \frac{\mathrm{d}}{\mathrm{d}t} i_f(t)$$

$$E_f(s) = R_f I_f(s) + L_f s I_f(s)$$

$$\frac{I_f(s)}{E_f(s)} = \frac{1}{R_f + L_f s}.$$

$$e_g(t) = K_g i_f(t)$$

$$E_g(s) = K_g I_f(s)$$

$$\frac{E_g(s)}{I_f(s)} = K_g.$$

$$e_a(t) = i_a(t)Z_l$$

$$E_a(s) = Z_l I_a(s)$$

$$\frac{E_a(s)}{I_a(s)} = Z_l.$$

$$e_{g}(t) = R_{a}i_{a}(t) + L_{a}\frac{d}{dt}i_{a}(t) + e_{a}(t)$$

$$E_{g}(s) = R_{a}I_{a}(s) + L_{a}sI_{a}(s) + E_{a}(s)$$

$$= (R_{a} + L_{a}s + Z_{l})I_{a}(s)$$

$$\frac{I_{a}(s)}{E_{g}(s)} = \frac{1}{L_{a}s + R_{a} + Z_{l}}.$$

Put everything together.

$$\frac{E_a(s)}{E_f(s)} = \frac{E_a(s)}{I_a(s)} \frac{I_a(s)}{E_g(s)} \frac{E_g(s)}{I_f(s)} \frac{I_f(s)}{E_f(s)} 
= \frac{K_g Z_l}{(L_f s + R_f) (L_a s + R_a + Z_l)}.$$

### **Block-diagram algebra**

